

**AMENDMENTS TO THE SPECIFICATION**

Following the title, please insert the following paragraphs:

**Cross-Reference to Prior Application**

This is a U.S. National Phase application under 35 U.S.C. §371 of International Patent Application No. PCT/JP2004/004814 filed April 1, 2004, and claims the benefit of Japanese Patent Application Nos. 2003-099059 filed April 2, 2003, 2003-099060 filed April 2, 2003 and 2003-099061 filed April 2, 2003, all of which are incorporated by reference herein. The International Application was published in Japanese on October 21, 2004 as WO 2004/090360 A1 under PCT Article 21(2).

Please replace the paragraph starting from page 18, line 7 with the following amended paragraph:

--The bearing having the above-mentioned structure is used in the state in which the bearing body 1 is impregnated with lubricating oil and the rotating shaft 2 is inserted into the bearing hole 3. Fig. 3 shows an example of a mechanism in which a rotating shaft 2 is supported at two points by the bearings. In the mechanism, a spiral gear 2a is formed on the circumferential surface of the rotating shaft 2, and both ends of the rotating shaft 2 are supported by the bearings. The mechanism is constructed so that a spiral gear 5-6 rotated by a driving device (not shown) is engaged with the spiral gear 2a of the rotating shaft 2 to rotate the rotating shaft 2. Although the rotating shaft 2 is actually not deflected as much as shown in Fig. 3, the rotating shaft 2 is exaggeratedly shown to clarify the point of the description.--

Please replace the paragraph starting from page 23, line 6 with the following amended paragraph:

--Next, as shown Fig. 5B, the first upper punch 11-12 is fitted into the hole 10a to strongly push down the sintered body W. The pushed-down sintered body W is interposed between the first upper punch 11-12 and the first lower punch 13 so as to be pressed from above and below by the first upper punch 11-12 and the first lower punch 13 to be slightly pushed and contracted. In addition, the outer surface of the sintered body W is pushed against the inner surface of the hole 10a to be corrected in a smooth cylindrical shape, and the outer surface of core rod 11 is pushed against the sintered body W to correct the inner surface of the sintered body W in a smooth cylindrical shape (a bearing hole 3, which includes a journal part 3a and has a constant diameter, is formed in the sintered body W).--

Please replace the paragraph starting from page 28, line 18 with the following amended paragraph:

--A bearing shown in Fig. 8 includes a bearing body 11 that is made of a sintered metal and has a bearing hole 13 formed therein. The bearing hole 13 has a circular cross section in a plane orthogonal to the longitudinal axis O of the rotating shaft 2, and is provided with a journal part 13a, first enlarged diameter parts 13b and second enlarged diameter parts 13c. The journal part 13a is provided roughly at the center of the bearing body 11 and has a diameter slightly larger than the diameter of the rotating shaft 12. In this case, the journal part 13a has a constant diameter at any position in the longitudinal direction thereof. Furthermore, the first enlarged diameter parts 13b are provided on longitudinal opposite sides of the journal part 13a so that the first enlarged diameter parts 13b are connected with both ends of the journal part 13a, respectively. In this case, the first enlarged diameter parts 13b are formed in a tapered shape having constant diameters to be enlarged toward the tips of the first enlarged diameter parts 13b. Moreover, the second enlarged diameter parts 13c are provided on longitudinal opposite sides (further outside of the first enlarged diameter parts 13b) of the first enlarged diameter parts 13b so that the second enlarged diameter parts 13c are connected with both ends of the first enlarged diameter parts 13b, respectively. In this case, the

second enlarged diameter parts 13c are formed in a tapered shape having constant diameters to be enlarged toward the tips of the second enlarged diameter parts 13c.--

Please replace the paragraph starting from page 32, line 12 with the following amended paragraph:

--In the bearing according to the present embodiment, the following usage is also considered. Even though the magnitude of torque transmitted to the rotating shaft 2 is constant in a case where the rotating shaft 2 is supported at plural points by the bearings, the oblique angles of the rotating shaft 2 are mutually different at the plural points as long as distances between the bearings and a mechanism (for example, the spiral gear 5-6 described in the first embodiment) for transmitting torque to the rotating shaft 2 are mutually different. In this case, although bearings having an enlarged diameter part matched with each of the oblique angles may be used separately, a plurality of bearings having different shapes should be prepared which increases cost. Accordingly, if a bearing having the enlarged diameter parts, which are formed in multiple steps so as to match each of the oblique angles, and is manufactured by employing the bearing according to the present invention, only one kind of bearing is used for supporting the rotating shaft 2 at plural points. For this reason, it is possible to reduce cost by standardizing parts.--

Please replace the paragraph starting from page 36, line 9 with the following amended paragraph:

--As shown in Figs. 12A to 12C, a die 30 including a hole 30a that has an inner diameter substantially equal to an outer diameter of a sintered body W, a fourth core rod 31 that has a round bar shape and can be inserted into the hole 30a from above with clearances, a third upper punch 32 that can be fitted into the hole 30a from above and has a simple annular tip surface, and a third lower punch 33 that can be fitted into the hole 30a from below and has a simple annular tip surface

are used in the process of forming the enlarged diameter part 3b on ~~both sides~~one side of the journal part 3a.--